

Combustion bowl in the crown of a piston for a diesel engine

The invention relates to a combustion bowl in the crown of a piston for a diesel engine, in accordance with the preamble of claim 1.

In a diesel engine with direct injection, combustion of the fuel/air mixture generally takes place in a combustion bowl that is formed into the crown of the piston of the diesel engine. It is known, in this connection, to provide the radially outer edge region of the combustion bowl with an undercut. A torque curve that is constant in a broad rpm range of the diesel engine in question was already mentioned as an advantage of this in the German patent document No. 721889. According to the German laying-open document 1055873, the cause of this lies in the swirling caused by the undercut, and an accompanying improvement in the evaporation of the fuel injected into the combustion bowl.

A piston for a diesel engine is known from the European patent document EP 105 933, having a combustion bowl disposed outside of the center, into the radially outer edge region of which fuel is injected by way of several injection openings or fuel nozzle. To improve the swirling and therefore the evaporation of the fuel, the piston has a plurality of recesses uniformly distributed over the edge region.

The cited state of the art has the disadvantage that because of the combustion of the fuel/air mixture, the radially outer edge of the combustion bowl, in particular, is exposed to very great temperature stress, which can result in damage of the combustion bowl.

In the German utility model DE 80 26 159, it is suggested to provide a cooling channel between the edge of a combustion bowl disposed in the center, and the piston crown edge, in order to solve this problem. If the piston known from the prior art last mentioned is supposed to be used in a two-valve engine, in which the fuel nozzles is disposed next to the inlet and outlet valve lying symmetrical to the piston axis, i.e. outside of the center relative to the longitudinal piston axis, it is advantageous if the combustion bowl is also disposed outside of the center relative to the longitudinal piston axis, so that the fuel can be injected into the combustion bowl by the fuel nozzle, without problems.

In this connection, there is the problem that there is no room for a cooling channel between the top land of the piston and the edge region of the combustion bowl disposed outside of the center that lies closest to the top land, so that at least in this

region, the cooling channel must be disposed farther away from the piston crown, in the boss region of the piston. This has the disadvantage that in this region, the edge of the combustion bowl, the bowl neck, and also the compression ring groove, which is subject to very great thermal stress, are cooled poorly and run the risk of being damaged. Another disadvantage in this connection is that the cooling channel, which furthermore lies very close to the piston crown, assumes a slanted position relative to the longitudinal piston axis, and this results in production technology problems.

Finally, the problem occurs, in this connection, that the distance between the part of the cooling channel disposed in the edge region of the piston crown and the edge region of the combustion bowl that is disposed outside of the center, which lies at a distance from the former, becomes so great that cooling of this edge region of the combustion bowl is also impaired.

It is the task of the invention to avoid the disadvantages of this state of the art. This task is accomplished with the characteristics contained in the characterizing part of the main claim. Practical embodiments of the invention are the object of the dependent claims.

Some exemplary embodiments of the invention will be described in the following, using the drawings. These show:

Fig. 1 a top view of a piston according to the invention, having a combustion bowl disposed outside of the center,

Fig. 2 a section through the crown region of the piston according to Figure 1, along the line II-II,

Fig. 3 a top view of a piston having a combustion bowl disposed outside of the center, which has an elliptically shaped bowl neck,

Fig. 4 a top view of a piston having a combustion bowl disposed outside of the center, which has no undercut on one side,

Fig. 5 a section through the crown region of the piston according to Figure 4, along the line V-V,

Fig. 6 a section through the crown region of the piston according to Figure 4, along the line VI-VI,

Fig. 7 a top view of the piston according to Figure 4, which has a hump-like molded-on part disposed in the center of the combustion bowl,

Fig. 8 a section through the crown region of the piston according to Figure 7, along the line VIII-VIII,

Fig. 9 a section through the crown region of the piston according to Figure 7, along the line IX-IX,

Fig. 10 a top view of a piston having a combustion bowl disposed outside of the center, which is composed of two regions that are semi-circular in shape, and which is undercut over its entire circumference,

Fig. 11 a top view of a piston having a combustion bowl disposed outside of the center, which is composed of two regions that are semi-circular in shape, and which has no undercuts on two sides that lie opposite one another,

Fig. 12 to 14 top views of pistons having combustion bowls disposed outside of the center, the shapes of which are adapted to the number and the orientation of the bores of injection nozzles.

Fig. 1 shows a top view of the crown 1 of a piston 2 for a diesel engine, into which a combustion bowl 4 having a radially outer delimitation that is configured to be unround is formed, disposed outside of the center, relative to the longitudinal piston axis 3, and having a circular bowl neck 5. Although the radially outer delimitations of the combustion bowls are not visible in the top views according to the following figures, they have not been drawn with a broken line, since the core of the present

invention relates, in particular, to the shape and arrangement of these bowls. However, the radially inner delimitation 6 of a cooled ring insert 7, which is shown in Fig. 2, a section through the crown region of the piston 2 according to Fig. 1, along the line II-II, is drawn with a broken line in Figure 1. Fig. 2 also shows the cross-sectional shape of the combustion bowl 4 with the bowl neck 5 and with the undercut 8 that surrounds the entire combustion bowl 4.

In this connection, the radially outer delimitation of the undercut 8 of the combustion bowl 4 comes so close to the radially inner delimitation of the cooled ring insert 7, in all regions, that not only the groove 10 for a compression ring not shown in the figure, but also all the edge regions of the combustion bowl 4 are cooled by the cooling channel 9.

In contrast to the combustion bowl according to Figures 1 and 2, the bowl neck 11 shown in Figure 3 is configured to be elliptical. The advantage of this lies in the fact that the undercut 8' that results therefrom is reduced in size close to the main axis region of the bowl neck 11, as compared with the exemplary embodiment shown in Figures 1 and 2.

The combustion bowl 12 shown in Figures 4 to 6 is disposed outside of the center relative to the longitudinal piston axis 3, in the region of the crown 13 of the piston 14, and is configured to be unround. As is shown, in particular, by the section according to Figure 6 through the crown region of the piston 14, the combustion bowl 12 has no undercut in the region 15, which brings about the best possible lowering of the temperature in the bowl edge and the bowl neck, due to the cooling effect of the cooling channel 9.

The configuration of the combustion bowl 16 according to Figures 7 to 9 has a molded-on part 17 configured in hump-like manner, disposed in the center, which assures better swirling and therefore an improvement in the combustion of the fuel sprayed into the combustion bowl 16.

The configurations of combustion bowls 18 to 22 shown in Figures 10 to 14 have shapes that are adapted to the orientation and the number of bores of the injection nozzles and the width of the fuel jets sprayed into the combustion bowls 18 to 22, proceeding from the nozzles. Thus, the combustion bowl 18 according to Figure 10 is disposed outside of the center relative to the longitudinal piston axis 3, provided with a circular bowl neck 26, and composed of two regions 23 and 24 that are arc-shaped in

cross-section and lie opposite one another. It has an undercut 38 over its entire circumference.

The combustion bowl 19 according to Figure 11 differs from the combustion bowl 18 (Fig. 10) in that it has no undercut at two regions 25 and 25' that lie opposite one another.

The combustion bowl 20 according to Fig. 12 differs from the combustion bowls 18 and 19 according to Fig. 10 and 11 in that the bowl neck is not disposed symmetrical to the combustion bowl 20 and has no undercut in the region 29. It is essentially composed of two circle segments 27 and 28.

The combustion bowls 21 and 22 according to Figures 13 and 14 consist of three circle segments, in each instance. However, it is also possible to compose the combustion bowls of more than three circle segments. In this connection, the size and arrangement of each circle segment can correspond to the direction and the width of the fuel jet introduced into the circle segment in question, whereby injection nozzles having bores of different diameters are used.

The combustion bowl 21 according to Figure 13 is formed from the three circle segments 30, 31, and 32, and a region 33 without undercut.

The combustion bowl 22 shown in Figure 14 consists of three circle segments 34, 35, 36 having different diameters, and of a circumferential undercut having a cross-section that varies over the circumference.

Other embodiments of combustion bowls, not shown in the figures, are characterized in that in this connection, not only the radially outer delimitations of the combustion bowls but also their bowl necks are configured to be unround. If it is taken into consideration in this connection that the production of the combustion bowls takes place in two or more machining steps with axis-symmetrical machining, in each instance, whereby the piston is displaced in the corresponding direction between the machining phases, it becomes clear that in this way, the possibility exists to give the combustion bowls essentially the shape of a triangle, a rectangle, or a polygon, with the exception of partially circle-shaped corner regions.

Reference Symbol List

1	crown of the piston 2
2, 2'	piston
3	longitudinal piston axis
4, 4'	combustion bowl
5	bowl neck
6	radially inner delimitation of the cooled ring insert 7
7	cooled ring insert
8, 8'	undercut
9	cooling channel
10	groove for a compression ring
11	elliptical bowl neck
12	combustion bowl
13	crown of the piston 14
14	piston
15	region of the combustion bowl 12
16	combustion bowl
17	molded-on part
18, 19, 20, 21, 22	combustion bowl
23, 24	region of the combustion bowl 18
25, 25'	region of the combustion bowl 19
26	bowl neck

